

Resonant grazing bifurcations in simple impacting systems

Indranil Ghosh and David J.W. Simpson

School of Mathematical and Computational Sciences
Massey University, Palmerston North, New Zealand

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Impact Oscillators

- ▶ Many engineering systems involve vibrations and impacts, e.g. impact print hammers, gear assemblies, machinery for milling, bells, and shock absorbers.

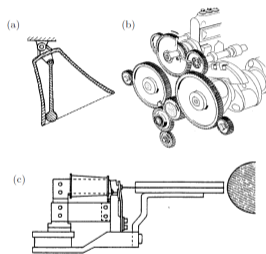


Figure: Examples of simple impacting systems: (a) a bell, (b) a gear assembly, (c) an impact print hammer. Picture taken from *di Bernardo, Champneys, Budd, Kowalczyk, 2008*.

The impact oscillator model

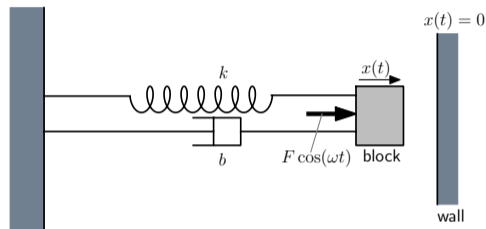


Figure: A hard-impact oscillator model: $\ddot{x} + b\dot{x} + kx + 1 = F \cos(\omega t)$ and $\dot{x} \mapsto -r\dot{x}$ whenever $x = 0$.

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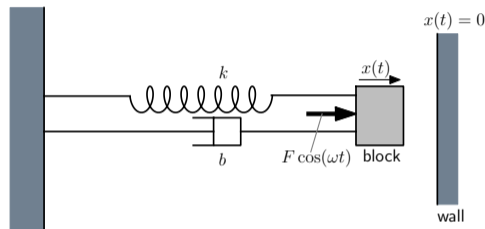


Figure: A hard-impact oscillator model: $\ddot{x} + b\dot{x} + x + 1 = F \cos(\omega t)$ and $\dot{x} \mapsto -r\dot{x}$ whenever $x = 0$.

- ▶ If the block hits the wall with zero velocity, this is a *grazing* impact.
- ▶ A grazing bifurcation occurs when the limit cycle has a grazing impact.

- ▶ S.W. Shaw, *J. Appl. Mech.*, 1985.
- ▶ S.W. Shaw and P.J. Holmes, *J. Sound Vib.*, 1983.
- ▶ J. de Weger *et al.*, *Phys. Rev. Lett.*, 1996.
- ▶ P. T. Piironen *et al.*, *J. Nonlinear Sci.*, 2004.
- ▶ J. Qiu and Z.C. Feng, *Computers & Structures*, 2000.
- ▶ T. Witelski *et al.*, *J. Sound Vib.*, 2014.
- ▶ S. Banerjee *et al.*, *Phys. Rev. E*, 2009.
- ▶ J. Molenaar *et al.*, *Nonlinearity*, 2001.
- ▶ E. Pavlovskaia *et al.*, *Int. J. Bifurcation Chaos*, 2010.

Experimental example

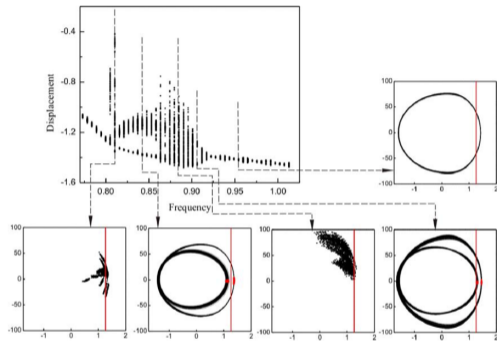


Figure: Bifurcation diagram obtained from the paper by Pavlovskaia *et al.*, Int. J. Bifurcation Chaos, 2010.

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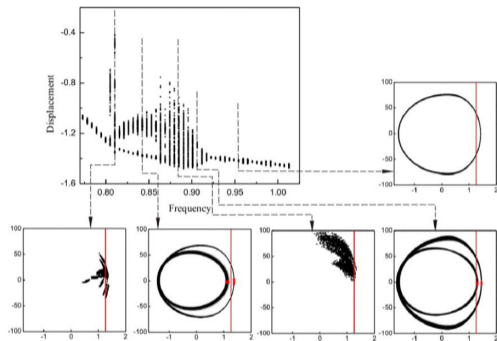


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- ▶ Why does a stable period-two solution appear so close to grazing?

- ▶ The nondimensionalised equations of our oscillator model are given by

$$\begin{aligned}\dot{x} &= y, \\ \dot{y} &= F \cos(\omega t) - by - x - 1,\end{aligned}$$

where $x(t)$ and $y(t)$ are the displacement and the velocity of the oscillator with the damping ratio $b > 0$.

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- ▶ We treat F as the primary bifurcation parameter.
- ▶ The values of F and t that occur at grazing are implicitly given by

$$\begin{aligned}t_{\text{graz}} &= \frac{1}{\omega} \tan^{-1} \left(\frac{b\omega}{1 - \omega^2} \right), \\ F_{\text{graz}}^2 &= (1 - \omega^2)^2 + b^2\omega^2.\end{aligned}$$

Typical phase portrait

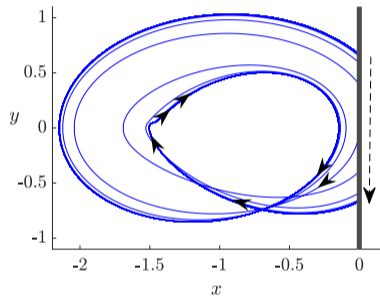


Figure: A typical phase portrait of the impact oscillator.

Bifurcation diagram

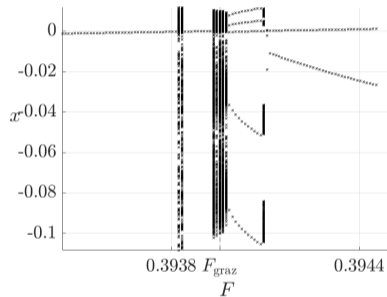


Figure: A typical bifurcation diagram of the impact oscillator.

Poincaré map

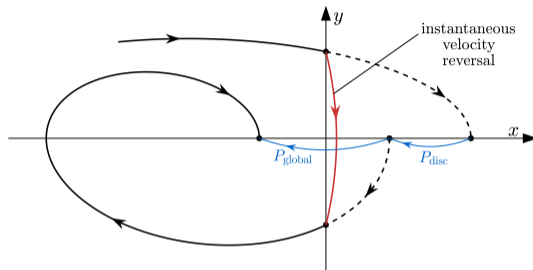


Figure: An illustration of the Poincaré map.

Poincaré map

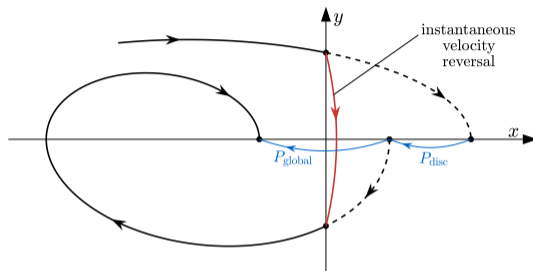


Figure: An illustration of the Poincaré map.

- ▶ We use $y = 0$ as the Poincaré section. The map is given by $(x', z') = P(x, z)$ where $z = t - t_{\text{graz}} \bmod \frac{2\pi}{\omega}$.
- ▶ We evaluate P numerically, using an explicit formula for the flow, and event detection for determining where orbits return to the Poincaré section.

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- ▶ Here

$$P_{\text{disc}}(x, z; F) = \begin{cases} \begin{bmatrix} x \\ z \end{bmatrix}, & x \leq 0, \\ \begin{bmatrix} r^2 x + \tilde{O}(3) \\ z - \frac{\sqrt{2}}{\omega}(1+r)\sqrt{x} + \tilde{O}(2) \end{bmatrix}, & x > 0. \end{cases}$$

Poincaré map

- ▶ To first order, the Taylor expansion of P_{global} about $(x, z; F) = (0, 0; F_{\text{graz}})$ can be written as

$$P_{\text{global}} = K \begin{bmatrix} x \\ z \end{bmatrix} + \frac{F - F_{\text{graz}}}{F_{\text{graz}}} \begin{bmatrix} 1 - a_{11} \\ -a_{21} \end{bmatrix} + O(2),$$

where

$$K = \begin{bmatrix} a_{11} & \omega^2 a_{12} \\ \frac{a_{21}}{\omega^2} & a_{22} \end{bmatrix},$$

and each a_{ij} is the (i, j) entry of

$$A = \exp\left(\frac{2\pi}{\omega} \begin{bmatrix} 0 & 1 \\ -1 & -b \end{bmatrix}\right).$$

Poincaré map

► Note that

$$a_{11} = \frac{\lambda_1 e^{\frac{2\pi}{\omega} \lambda_2} - \lambda_2 e^{\frac{2\pi}{\omega} \lambda_2}}{\lambda_1 - \lambda_2}, \quad a_{12} = \frac{e^{\frac{2\pi}{\omega} \lambda_1} - e^{\frac{2\pi}{\omega} \lambda_2}}{\lambda_1 - \lambda_2},$$
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- ▶ Here,

$$\alpha = -\frac{b}{2}, \quad \beta = \sqrt{1 - \frac{b^2}{4}}.$$

- ▶ Also K has trace $\tau = 2e^{\frac{2\pi\alpha}{\omega}} \cos\left(\frac{2\pi\beta}{\omega}\right)$ and determinant $\delta = e^{\frac{4\pi\alpha}{\omega}}$.

Numerics

- ▶ For a period- p solution of our map P with one point in $x > 0$, this point is a fixed point of $P_{\text{global}}^p \circ P_{\text{disc},R}$.

Numerics

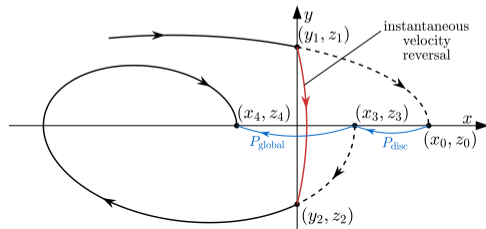
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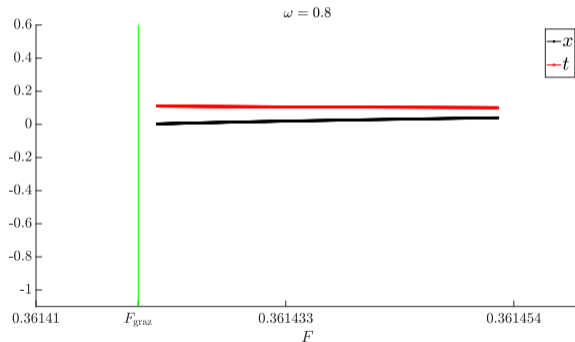
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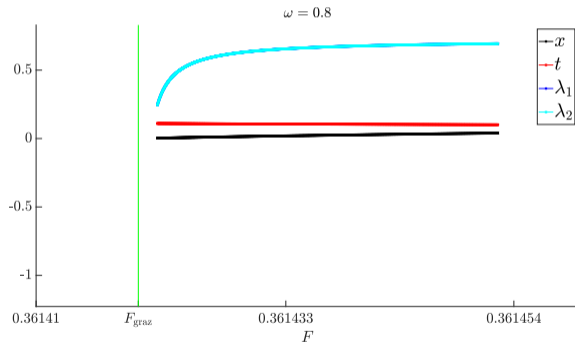
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- ▶ Since $P_{\text{global}}^p \circ P_{\text{disc},R}$ is smooth, standard numerical methods like Newton's method can be used to follow fixed points while $x > 0$.
- ▶ That is, given a guess for (x_0, z_0) , we compute (y_1, z_1) , (y_2, z_2) , and (x_3, z_3) , and $(x_4, z_4) = P_{\text{global}}^p(x_3, z_3; F)$. Then let $G(x_0, z_0; F) = (x_4, z_4) - (x_0, z_0)$ and continue zeros of G .



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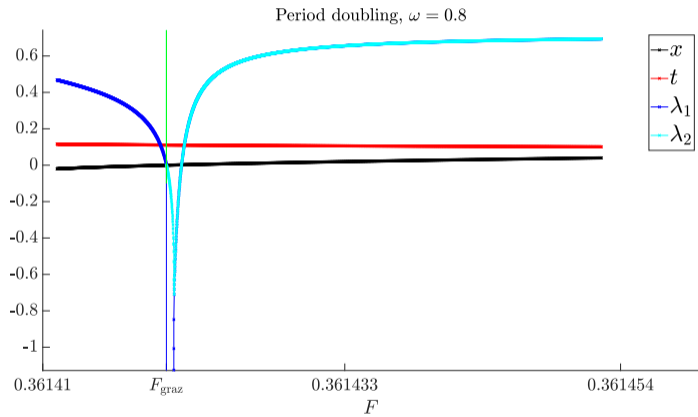


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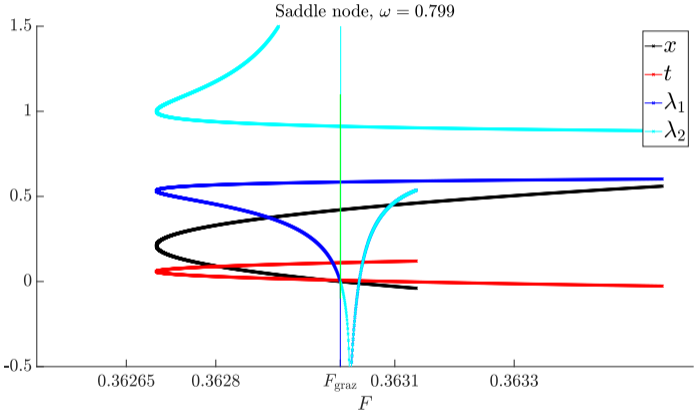
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- ▶ The function V maps the impact velocity and z -value to the variation (or change) in displacement and z -value.
- ▶ We call the function V as the VIVID function that follows the zeros of a function mapping **V**elocity Into **V**ariation In **D**isplacement.

One-parameter bifurcation diagram



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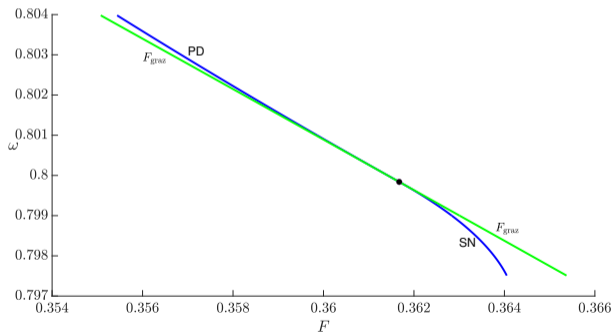


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- ▶ The location of the codimension-two point is understood.



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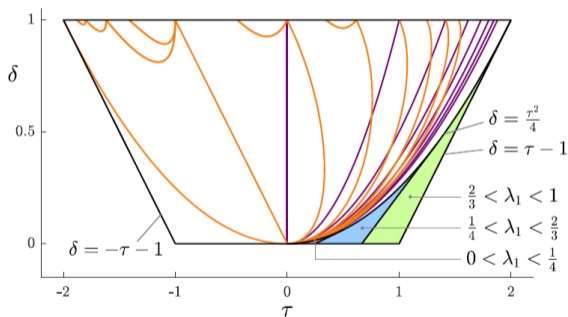


Figure: Division of the (τ, δ) plane.

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- ▶ Solving which we get a rational ratio of ω and β given by

$$\frac{\omega}{\beta} = \frac{4}{5}$$

corresponding to *resonance*.

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- ▶ We produce two-parameter bifurcation diagrams showing curves of saddle-node and period-doubling bifurcation emanating from a codimension-two grazing bifurcation.
- ▶ However, it remains to unfold such codimension-two points theoretically (and we have started to work on this). Hopefully, this can explain why the SN curve bends away from F_{graz} faster than the PD curve.

Acknowledgements



Thank you! Questions?